

BMA4723 VEHICLE DYNAMICS

Ch4 Vehicle Equation of Motions

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Chapter Description

Aims

- Explain the coordinates axes of the vehicle
- Explain the velocity and acceleration vector of the vehicle from the vector unit
- Explain the side-slip angle of the vehicle

Expected Outcomes

- Students are able to derive the equations of motion based on the unit vector.
- Students are able to determine the acceleration and deceleration of the vehicle by using the equations of motion.
- Students are able to know how side-slip angle of the vehicle can be generated.

References

- M.Abe, Vehicle Handling Dynamics Theory and Application, Second Edition, Published by Elsevier Ltd, 2015
- Thomas D.Gillespie, Fundamental of Vehicle Dynamics, Published by Society of Automotive Engineers





- 4.1 Velocity and Acceleration of the Vehicle
- 4.2 Side-slip Angle of the Vehicle



- In the analysis of the velocity and acceleration of the vehicle, the fixed coordinate axes at the vehicle is used.
- Fig.1 shows the fixed coordinate axes at the vehicle.



Figure 1 Fixed coordinate at the vehicle

From Fig.1:

- *P*: center gravity of the vehicle.
- *R*: position vector from the fixed coordinate on the ground to the fixed coordinate at the vehicle.
- *u*: the longitudinal velocity of point *P*at the *x*-axis
- *v*: the lateral velocity of point *P* at the *y*-axis
- β : side slip angles of the vehicle
- *r*: yaw rate of the vehicle



- Based on Fig.1, the position vector of the vehicle from the reference on the ground (coordinate system X − Y), is defined as R.
- Then, the velocity vector, \dot{R} can be written as:

$$\dot{R} = ui + vj$$
 (Eq.1)

where *i* and *j* are the unit vectors in *x* and *y* directions respectively.



• Differentiating the velocity \dot{R} in Eq.1 with time, the acceleration of the vehicle \ddot{R} can be written as:

$$\ddot{R} = \dot{u}i + u\dot{i} + \dot{v}j + v\dot{j} \qquad (Eq.2)$$



 Now, considered that the unit vectors at the fixed coordinate on the ground as i_F and j_F.





Figure 2 Unit vectors at the ground and at the vehicle

 From Fig.2, the relation of the unit vector at the x and y axis (i and j), to the reference X and Y axis on the ground (i_F and j_F) can be expressed as:

$$i = cos\theta i_f + sin\theta j_f$$
 (Eq.3)
 $j = -sin\theta i_f + cos\theta j_f$ (Eq.4)



Then, differentiate *i* and *j* with the time:

$$i = -\dot{\theta} \sin\theta i_F + \dot{\theta} \cos\theta j_F = r(-\sin\theta i_F + \cos\theta j_F) = rj$$

$$j = -\dot{\theta} \cos\theta i_F - \dot{\theta} \sin\theta j_F = -r(\cos\theta i_F + \sin\theta j_F) = -ri$$
(Eq.6)

• Substitute Eq.5 and Eq.6 into Eq.2:

$$\ddot{R} = \dot{u}i + u\tilde{i} + \dot{v}j + v\tilde{j}$$
$$\ddot{R} = \dot{u}i + u(rj) + \dot{v}j + v(-ri)$$
$$\ddot{R} = (\dot{u} - vr)i + (\dot{v} + ur)j$$
(Eq.7)

• From Eq.7, the longitudinal and lateral accelerations of the vehicle at point *P* can be expressed as:

$$a_x = \dot{u} - vr \tag{Eq.8}$$

$$a_{\nu} = \dot{\nu} - ur \tag{Eq.9}$$



The longitudinal and lateral acceleration of the vehicle are illustrated in Fig.3.





Figure 3 Longitudinal and lateral acceleration of the vehicle

• From the translational system of Newton's Second Law:

$$\sum F_x = ma_x$$
(Eq.10)
$$\sum F_y = ma_y$$
(Eq.11)

- The acceleration of the vehicle at the x and y axis can be expressed from the Eq.8 and Eq.9.
- Then, Eq.10 and Eq.11 can be written as:

$$m\left(\frac{du}{dt} - vr\right) = {}^{x}F_{FR} + {}^{x}F_{FL} + {}^{x}F_{RR} + {}^{x}F_{RL}$$
(Eq.12)

$$m\left(\frac{dv}{dt} - vr\right) = {}^{y}F_{FR} + {}^{y}F_{FL} + {}^{y}F_{RR} + {}^{y}F_{RL}$$
(Eq.13)

Where:

 ${}^{x}F$ is the traction or braking force and ${}^{y}F$ is the lateral force.



- In general, the vehicle travelling direction is same as the vehicle longitudinal direction.
- At the certain condition such as during cornering, the angle between the vehicle traveling direction and longitudinal direction is not same.
- The angle created between the vehicle travelling direction and longitudinal direction is called as side-slip angle, β .



Figure 4 Side slip angle of the vehicle



• From Fig.4, the equation of side-slip angle, β is:

$$tan^{-1} = (v/u)$$
 (Eq.14)

• Then, the velocity and acceleration at the longitudinal and lateral axes are:

$$u = V \cos\beta$$
 (Eq.15)
 $\dot{u} = -V \sin\beta\dot{\beta}$ (Eq.16)

$$v = V \sin\beta$$
 (Eq.17)
 $\dot{v} = V \cos\beta\dot{\beta}$ (Eq.18)



- In the case of β is very small, $cos\beta \cong 1$ and $sin\beta \cong \beta$.
- At that moment, Eq.15 to Eq.18 can be expressed as:

$$u = V \cos\beta \cong V$$
 (Eq.19)
 $\dot{u} = -V \sin\beta\dot{\beta} \cong -V\beta\dot{\beta}$ (Eq.20)

$$v = V \sin\beta \cong V\beta$$
 (Eq21)
 $\dot{v} = V \cos\beta\dot{\beta} \cong V\dot{\beta}$ (Eq22)



• Substitutes Eq.19 to Eq.22 into Eq1 and Eq.7:

$$\dot{R} = ui + vj$$
 (Eq.1)

$$\dot{R} = Vi + V\beta j \tag{Eq.23}$$

$$\ddot{R} = (\dot{u} - vr)i + (\dot{v} + ur)j$$
 (Eq.7)

$$\ddot{R} = \left(-V\beta\dot{\beta} - V\beta r\right)i + \left(V\dot{\beta} + Vr\right)j$$

$$\ddot{R} = -V\beta(\dot{\beta} + r)i + V(\dot{\beta} + r)j$$
(Eq.24)



Fig.5 shows how the side-slip angle effect the velocity and acceleration at the longitudinal and lateral axes



Figure 5 Side-slip angle effect the velocity and acceleration at the longitudinal and lateral axes



Conclusion of The Chapter 4

- Conclusion #1
 - The fixed coordinate of the vehicle can be used to analyse the velocity and acceleration of the vehicle.
- Conclusion #2
 - The side-slip angle of the vehicle is the angle between the vehicle travelling direction and longitudinal direction.





Vehicle Dynamics

Chapter 4

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