## ENGINEERING MECHANICS BAA1113

Chapter 4: Force System Resultants (Static)
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## Chapter Description

- Aims
- To explain the Moment of Force (2D-scalar formulation \& 3D-Vector formulation)
- To explain the Principle Moment
- To explain the Moment of a Couple
- To explain the Simplification of a Force and Couple System
- To explain the Reduction of Simple Distributed Loading
- Expected Outcomes
- Able to solve the problems of MOF and COM in the mechanics applications by using principle of moments
- References
- Russel C. Hibbeler. Engineering Mechanics: Statics \& Dynamics, $14^{\text {th }}$ Edition


## Chapter Outline

4.1 Moment of Force (MOF) -Part I
4.2 Principle of Moment -Part II
4.3 Moment of Couple (MOC) Part III
4.4 Simplification of a Force and Couple System
4.5 Reduction of Simple Distributed Loading- part IV


### 4.5 Reduction of a simple distributed <br> What is loading

distributed loading?

-In many situations, a surface area of a body is subjected to a distributed load

- Such forces are caused by winds, fluids, or the weight of items on the body's surface
-Distributed loadings are defined by using a loading function $w=w(x)$ that indicates the intensity of the loading along the length of the member
-Intensity is measured in $\mathrm{N} / \mathrm{m}$
-The external effects caused by a coplanar distributed load acting on a body can be represented by a single resultant force
-The magnitude of the resultant force is equal to the total area under the distributed loading diagram $w=w$ ( $x$ )
-The location of the resultant force is given by the fact that its line of action passes through the centroid or geometric center of this area


## Magnitude of resultant force



Consider an element of length dx.
The force magnitude dF acting on it is given as

$$
\mathrm{dF}=\mathrm{w}(\mathrm{x}) \mathrm{dx}
$$

The net force on the beam is given by
$+\downarrow \mathrm{F}_{\mathrm{R}}=\int_{\mathrm{L}} \mathrm{dF}=\int_{\mathrm{L}} \mathrm{w}(\mathrm{x}) \mathrm{dx}=\mathrm{A}$
Here A is the area under the loading curve $\mathrm{w}(\mathrm{x})$.

## Location of the resultant force



The force dF will produce a moment of $(\mathrm{x})(\mathrm{dF})$ about point O .


Assuming that $\mathrm{F}_{\mathrm{R}}$ acts at ${ }^{\bar{x}}$, it will produce the moment about point O as
$\Gamma_{+} \mathrm{M}_{\mathrm{RO}}=\left(^{\bar{x}}\right)\left(\mathrm{F}_{\mathrm{R}}\right)={ }^{\bar{x}} \int_{\mathrm{L}} \mathrm{w}(\mathrm{x}) \mathrm{dx}$

## Location of the resultant force



Comparing the last two equations, we get

$$
\bar{x}=\frac{\int_{L} x w(x) d x}{\int_{L} w(x) d x}=\frac{\int_{A} x d A}{\int_{A} d A}
$$

You will learn more detail later, but $F_{R}$ acts through a point " $C$," which is called the geometric center or centroid of the area under the loading curve $w(x)$.

## Example 4.23

Determine the concentrated loads (which is a common name for the resultant of the distributed load)


The rectangular load: find the area of rectangular
$\mathrm{F}_{\mathrm{R}}=400 \times 10=4,000 \mathrm{lb}$
$\bar{x}=\frac{10}{2}=5 \mathrm{ft}$ location is the centroid of rectangular

## Example 4.23

Determine the concentrated loads (which is a common name for the resultant of the distributed load)


The triangular loading: find the area of triangular
$\mathrm{F}_{\mathrm{R}}=\frac{1}{2}(600)(6)=1,800 \mathrm{~N}$ and $\bar{x}=6-(1 / 3) 6=4 \mathrm{~m}$
Please note that the centroid of a right triangle is at a distance one third the width of the triangle as measured from its base

## Example 4.24

The loading on the beam as shown. Determine equivalent force and its location from point A.


1) The distributed loading can be divided into two parts. (one rectangular loading and one triangular loading).
2) Find $F_{R}$ and its location for each of the distributed loads.
3) Determine the overall $\mathrm{F}_{\mathrm{R}}$ of the point loadings and its location.

## Solution Example 4.24



For the triangular loading of height $150 \mathrm{lb} / \mathrm{ft}$ and width 6 ft , $\mathrm{F}_{\mathrm{R} 1}=(0.5)(150)(6)=450 \mathrm{lb}$
and its line of action is at $\bar{x}_{1}=(2 / 3)(6)=4 \mathrm{ft}$ from A

For the rectangular loading of height $150 \mathrm{lb} / \mathrm{ft}$ and width 8 ft , $\mathrm{F}_{\mathrm{R} 2}=(150)(8)=1200 \mathrm{lb}$ and its line of action is at $\bar{x}_{2}=6+(1 / 2)(8)=10 \mathrm{ft}$ from A

## Solution Example 4.24



The equivalent force and couple moment at A will be

$$
\begin{gathered}
\mathrm{F}_{\mathrm{R}}=450+1200=1650 \mathrm{lb} \\
+\left(\mathrm{M}_{\mathrm{RA}}=4(450)+10(1200)=13800 \mathrm{lb} \cdot \mathrm{ft}\right.
\end{gathered}
$$

Since ( $\mathrm{F}_{\mathrm{R}} \bar{x}$ ) has to equal $\mathrm{M}_{\mathrm{RA}}$ : $1650 \bar{x}=13800$
Solve for $\bar{x}$ to find the equivalent force's location. $\bar{x}=8.36 \mathrm{ft}$ from A .

## Example 4.25

The loading on the beam as shown. Determine equivalent force and couple moment acting at point O


1) The distributed loading can be divided into two parts-two triangular loads
2) Find $F_{R}$ and its location for each of these distributed loads
3) Determine the overall $F_{R}$ of the point loadings and couple moment at point $O$

## Solution Example 4.25



For the triangular loading(right) of height $6 \mathrm{kN} / \mathrm{m}$ and width 7.5 m , $\mathrm{F}_{\mathrm{R} 1}=(0.5)(6)(7.5)=22.5 \mathrm{kN}$
and its line of action is at $\bar{x}_{1}=(2 / 3)(7.5)=5 \mathrm{~m}$ from O
For the triangular loading (left) of height $6 \mathrm{kN} / \mathrm{m}$ and width 4.5 m ,
$\mathrm{F}_{\mathrm{R} 2}=(0.5)(6)(4.5)=13.5 \mathrm{kN}$ and its line of action is at $\bar{x}_{2}=7.5+(1 / 3)(4.5)=9 \mathrm{~m}$ from O

## Solution Example 4.25



For the combined loading of the three forces, add them.

$$
F_{R}=22.5+13.5+15=51 \mathrm{kN}
$$

The couple moment at point $O$ will be

$$
+\left(\mathrm{M}_{\mathrm{RO}}=500+5(22.5)+9(13.5)+12(15)=914 \mathrm{kN} \cdot \mathrm{~m}\right.
$$

## Conclusion of The Chapter 4

- Conclusions
- The reduction of force simple loading has been identified
- The external effects caused by a coplanar distributed load acting on a body can be represented by a single resultant force
- The reduction of force analysis have been implemented to solve resultant force and moment problems in specified axis



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