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# THEORY OF STRUCTURES

## CHAPTER 2 : DEFLECTION (UNIT LOAD METHOD)

### PART 2

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by Saffuan Wan Ahmad

# Chapter 2 : Part 2 – Unit Load Method

- Aims
  - Determine the slope and deflection by using Unit Load Method
- Expected Outcomes :
  - Able to analyze determinate beam – deflection and slope by Unit Load Method
- References
  - Mechanics of Materials, R.C. Hibbeler, 7th Edition, Prentice Hall
  - Structural Analysis, Hibbeler, 7th Edition, Prentice Hall
  - Structural Analysis, SI Edition by Aslam Kassimali, Cengage Learning
  - Structural Analysis, Coates, Coatie and Kong
  - Structural Analysis - A Classical and Matrix Approach, Jack C. McCormac and James K. Nelson, Jr., 4th Edition, John Wiley

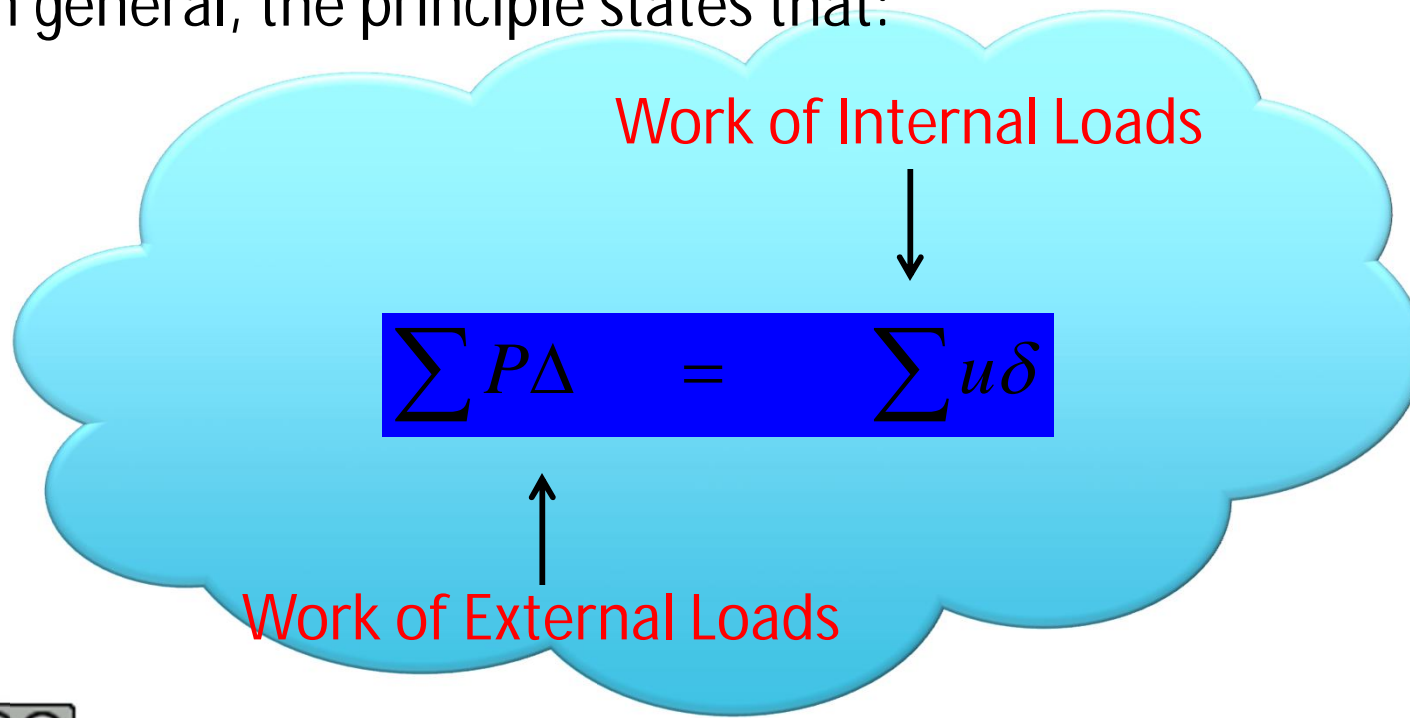


## Principle of Virtual Work

- The internal work in transversely loaded beams is taken equal to the strain energy due to bending moment
- The virtual force  $F_i$  in the  $i^{\text{th}}$  mass element in  $\Delta = F_i^* e_i$  may be taken equal to the bending moment  $m_{ij}$  in the  $i^{\text{th}}$  mass element due to a unit load at coordinate  $j$

# Principle of Virtual Work (Displacement)

- Sometimes referred as the Unit-Load Method
- Generally provides of obtaining the displacement and slope at a specific point on structure i.e. beam, frame or truss
- In general, the principle states that:



# Principle of Virtual Work (Displacement)

- Consider the structure (or body) to be of arbitrary shape
- Suppose it is necessary to determine the displacement  $\Delta$  of point A on the body caused by the “real loads”  $P_1$ ,  $P_2$  and  $P_3$

# Principle of Virtual Work (Displacement)

- Since no external load acts on the body at A and in the direction of the displacement  $\Delta$ , the displacement can be determined by first placing on the body a “virtual” load  $P'$  that this force  $P'$  acts in the same direction as  $\Delta$ , (see Figure)

# Principle of Virtual Work (Displacement)

- We will choose  $P'$  to have a unit magnitude,  $P' = 1$
- Once the virtual loadings are applied, then the body is subjected to the real loads  $P_1$ ,  $P_2$  and  $P_3$ , (see Figure)
- Point A will be displaced an amount  $\Delta$  causing the element to deform an amount  $dL$

# Principle of Virtual Work (Displacement)

- As a result, the external virtual force  $P'$  & internal load  $u$  "ride along" by  $\Delta$  and  $dL$  and therefore, perform external virtual work of  $1 \cdot \Delta$  on the body and internal virtual work of  $u \cdot dL$  on the element

$$1 \cdot \Delta = \sum u \cdot dL$$

- By choosing  $P' = 1$ , it can be seen from the solution for  $\Delta$  follows directly since  $\Delta = \sum u dL$



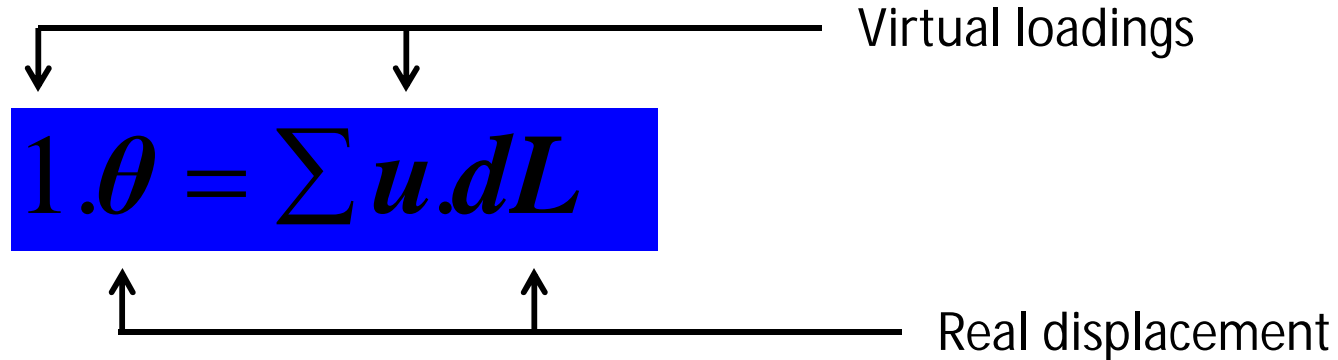
# Principle of Virtual Work (Slope)

- A virtual couple moment  $M'$  having a unit magnitude is applied at this point
- This couple moment causes a virtual load  $u_\theta$  in one of the elements of the body

# Principle of Virtual Work (Slope)

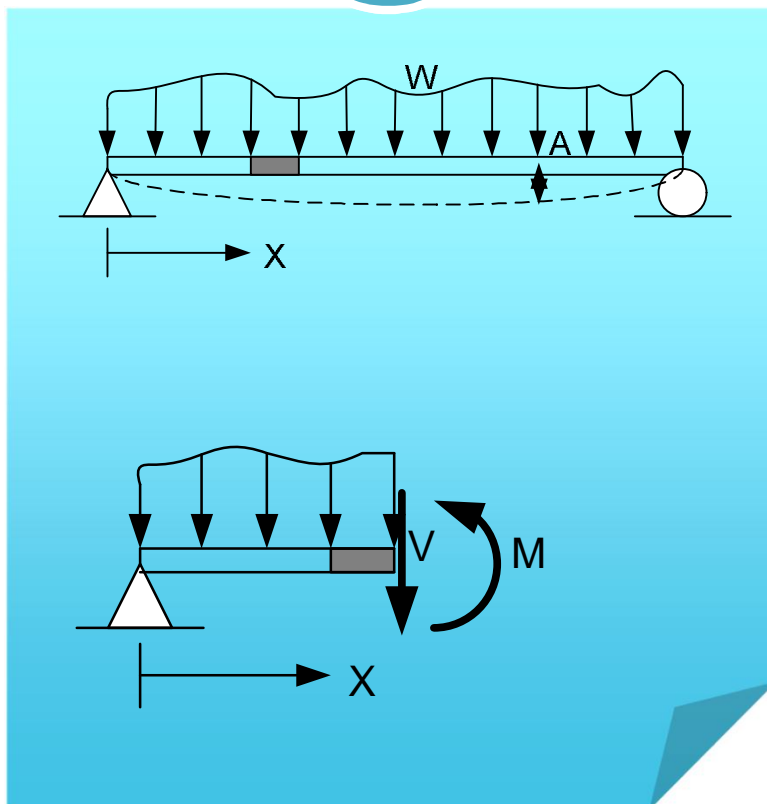
- Assuming that the real loads deform the element an amount  $dL$ , the rotation  $\theta$  can be found from the virtual-work equation

# Principle of Virtual Work (Slope)

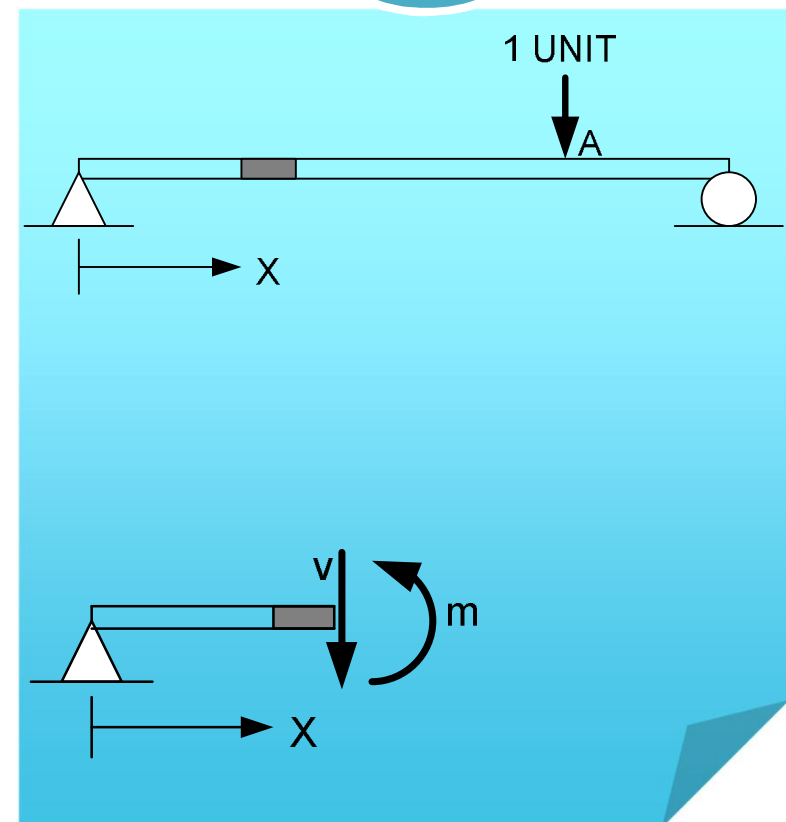


# PRINCIPLE OF UNIT LOAD METHOD

## REAL LOAD



## VIRTUAL LOAD



- The element deform or rotate  $d\theta = (M / EI) dx$
- The external virtual work done by the unit load is  $1 \cdot \Delta$
- The internal virtual work done by the moment,  $m$

$$m d\theta = m(M/EI) dx$$

$$1 \cdot \Delta = \int_0^L \frac{mM}{EI} dx$$

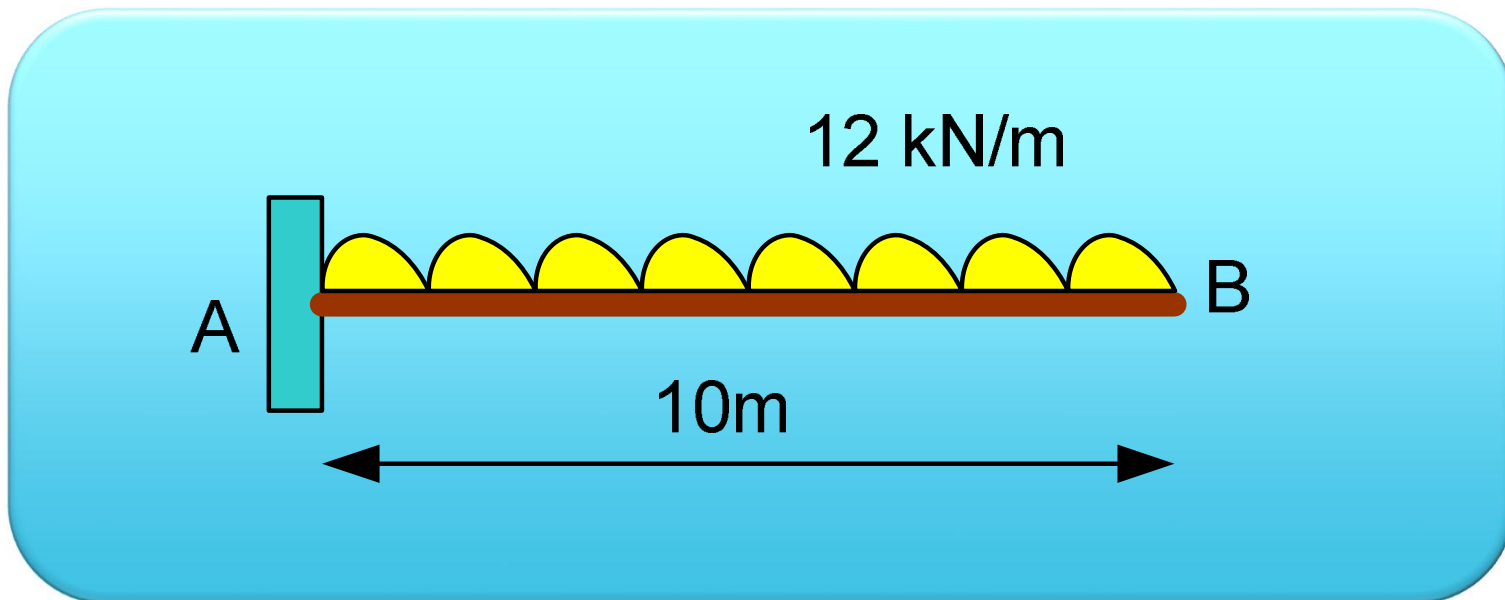
**Similarly**

$$1 \cdot \theta = \int_0^L \frac{mM}{EI} dx$$

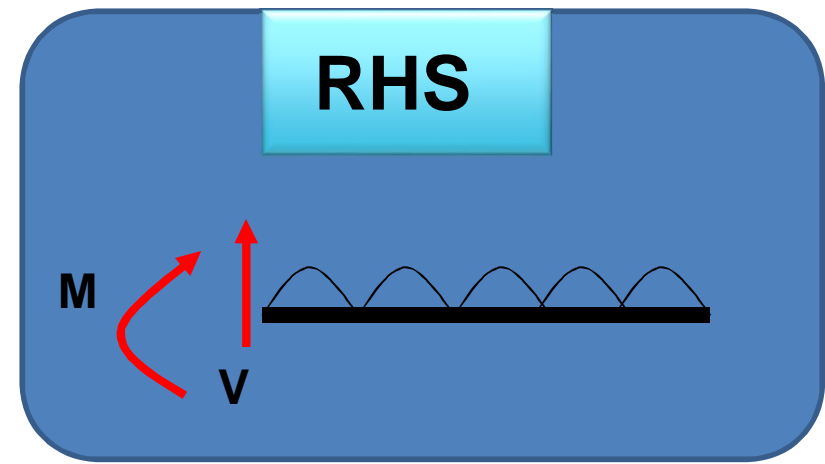
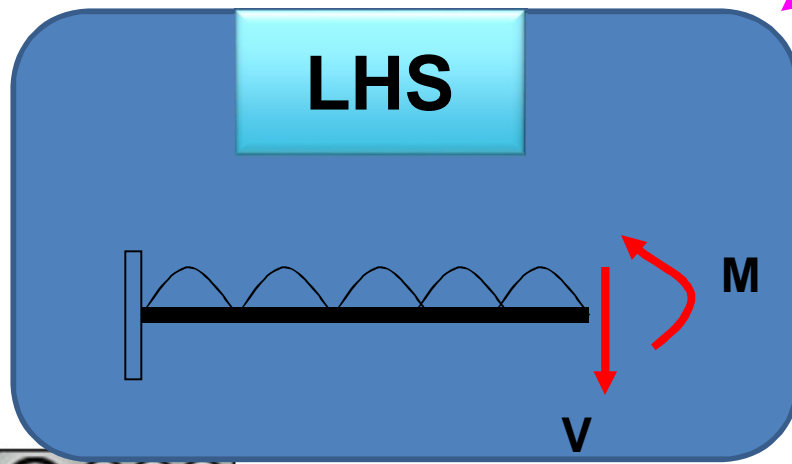
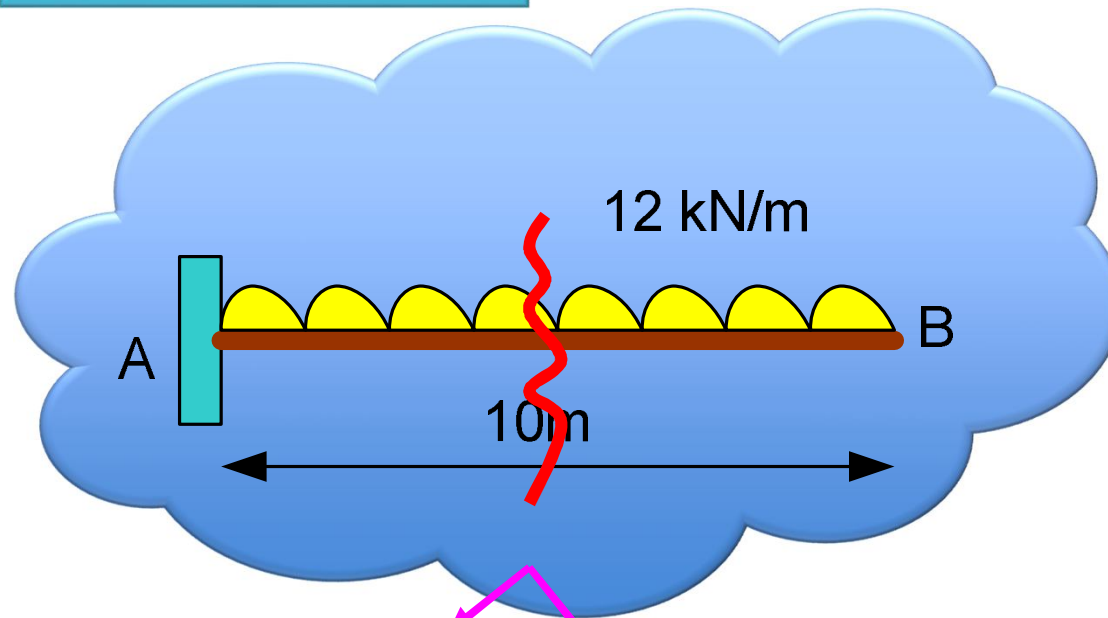
# Example 1

Determine the displacement at point B of the steel beam shown in figure.

Take  $E = 200\text{GPa}$ ,  $I = 500 \times 10^6 \text{ mm}^4$

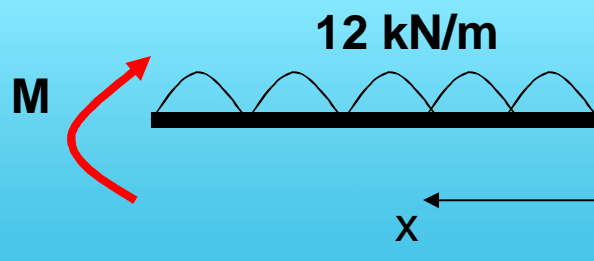


## Real Moment, $M$



## Real Moment, M

RHS



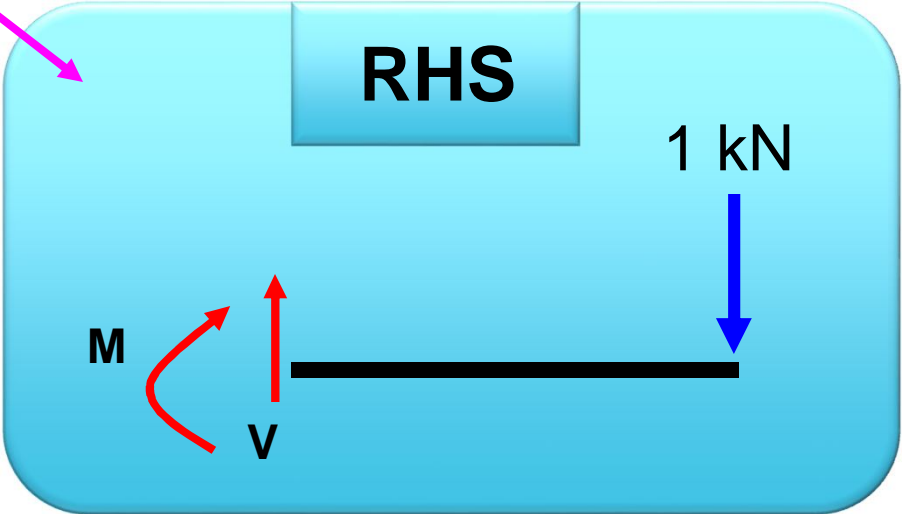
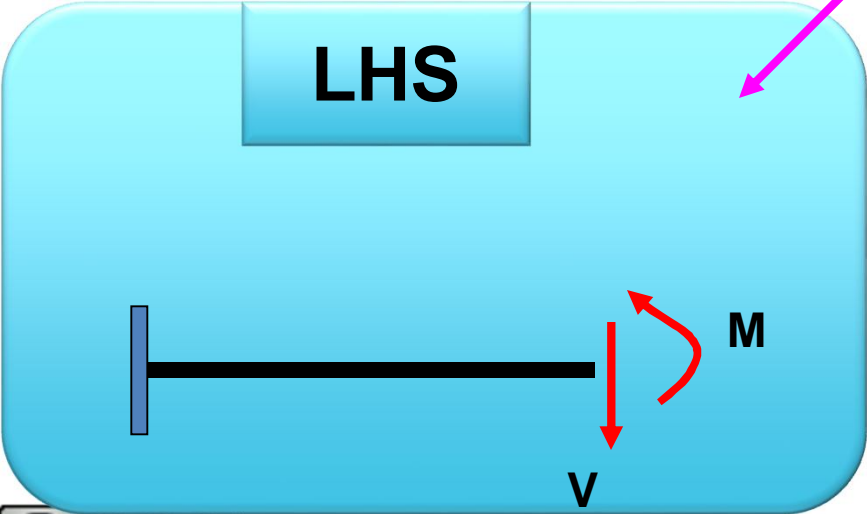
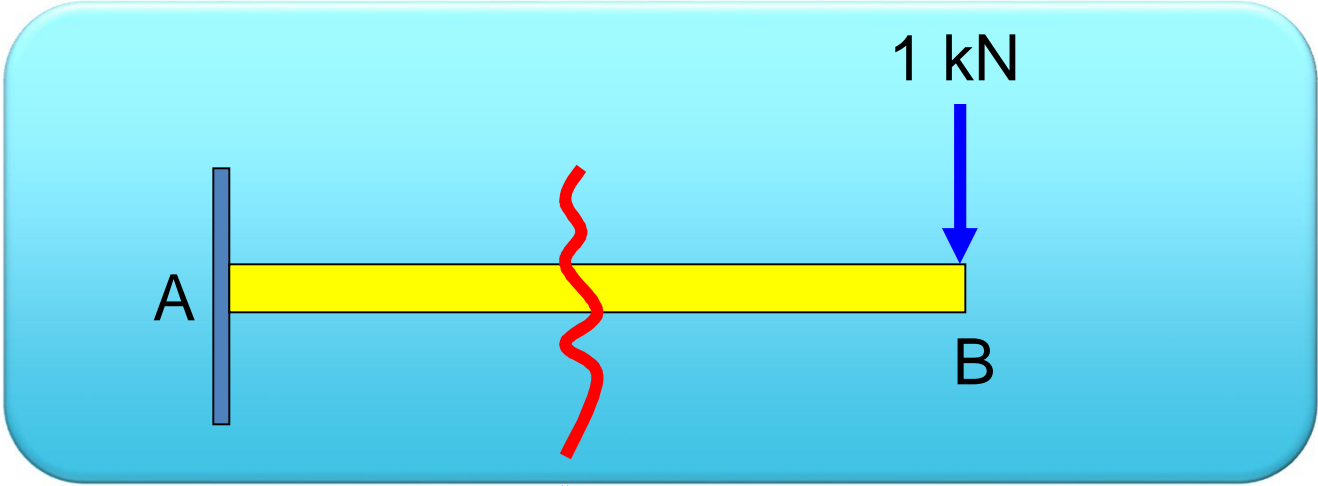
$$\sum M_x = 0 \text{ (clockwise + ve)}$$

$$M + \frac{12x^2}{2} = 0$$

$$M = -6x^2$$



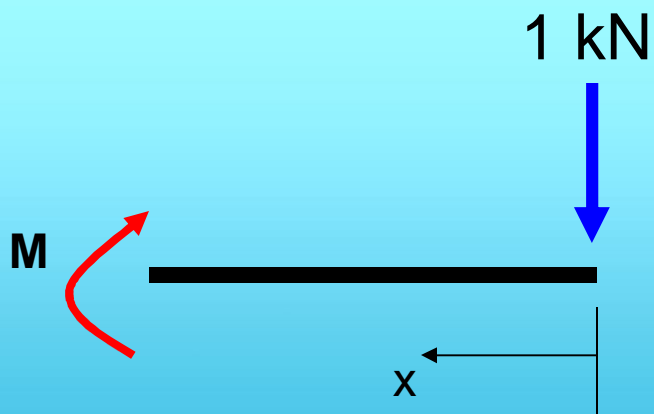
# Virtual Moment, $m$



Virtual Moment,  $m$

Considered RHS

$0 < x < 10$



$$\sum M_x = 0 \text{ (clockwise + ve)}$$

$$m + 1 \cdot x = 0$$

$$m = -1 \cdot x$$

## Virtual-Work Equation

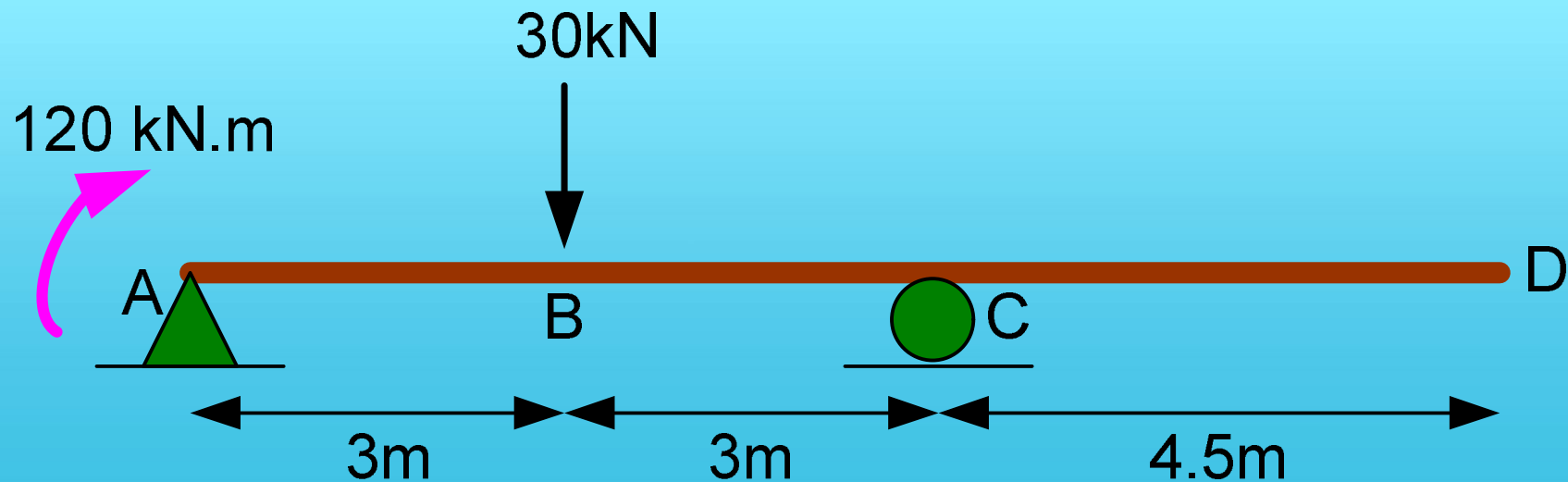
$$\begin{aligned}
 1 \text{ kN} \cdot \Delta &= \int_0^L \frac{mM}{EI} dx \\
 &= \int_0^{10} \frac{(-x)(-6x^2)}{EI} dx \\
 &= \frac{15 \times 10^3 \text{ kN} \cdot \text{m}^3}{EI}
 \end{aligned}$$

OR

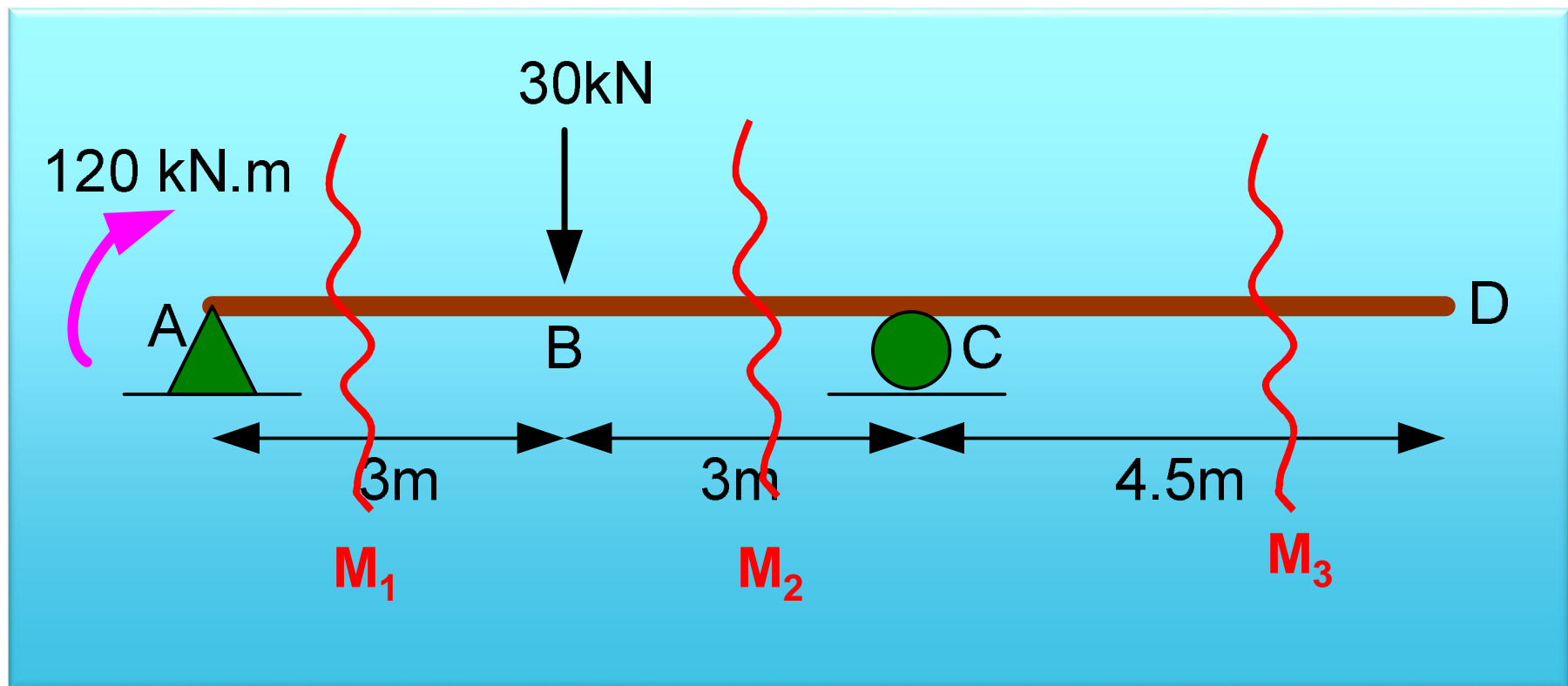
$$\begin{aligned}
 \Delta_B &= \frac{15 \times 10^3 \text{ kN} \cdot \text{m}^3}{200(10^6) \text{ kN} / \text{m}^2 (500(10^6) \text{ mm}^4) (10^{-12} \text{ m}^4 / \text{mm}^4)} \\
 &= 0.150 \text{ m} = 150 \text{ mm}
 \end{aligned}$$

## Example 2

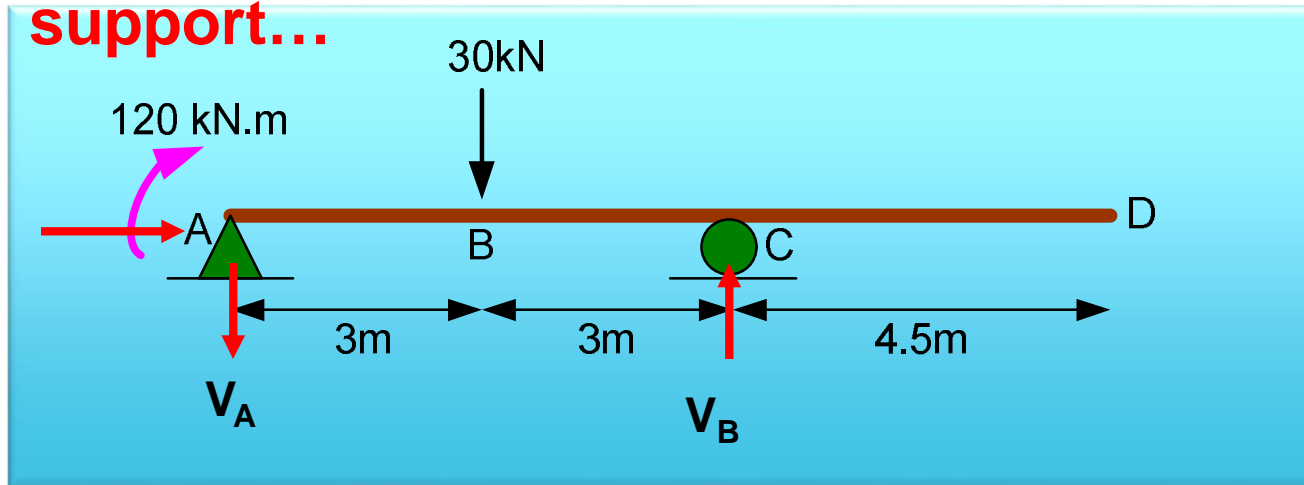
Determine the displacement at D of the steel beam in figure. Take  $E = 200\text{GPa}$ ,  $I = 300\text{E}6 \text{ mm}^4$



## Real Moment, $M$



## Determine the reaction at support...



$$\sum M_A = 0 \text{ (clockwise + ve)}$$

$$120 + 30(3) - V_B(6) = 0$$

$$V_B = 35 \text{ kN}$$

$$\sum F_y = 0 \text{ (upward + ve)}$$

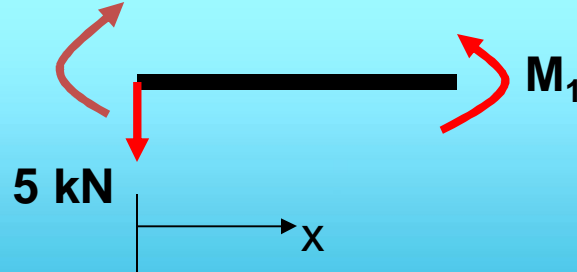
$$-V_A + 35 - 30 = 0$$

$$V_A = 5 \text{ kN}$$

Member AB : LHS

$$0 \leq x \leq 3$$

120 kN.m



$$\sum M_x = 0 \text{ (clockwise + ve)}$$

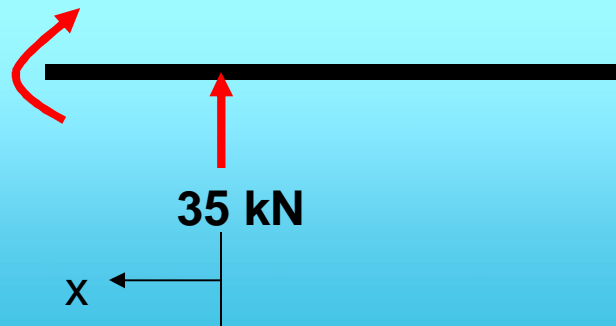
$$-M_1 - 5(x) + 120 = 0$$

$$M_1 = 120 - 5x$$

Member BC : RHS

$$0 \leq x \leq 3$$

$M_2$



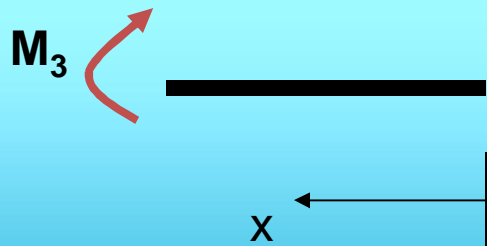
$$\sum M_x = 0 \text{ (clockwise + ve)}$$

$$M_2 - 35(x) = 0$$

$$M_2 = 35(x)$$

Member CD : RHS

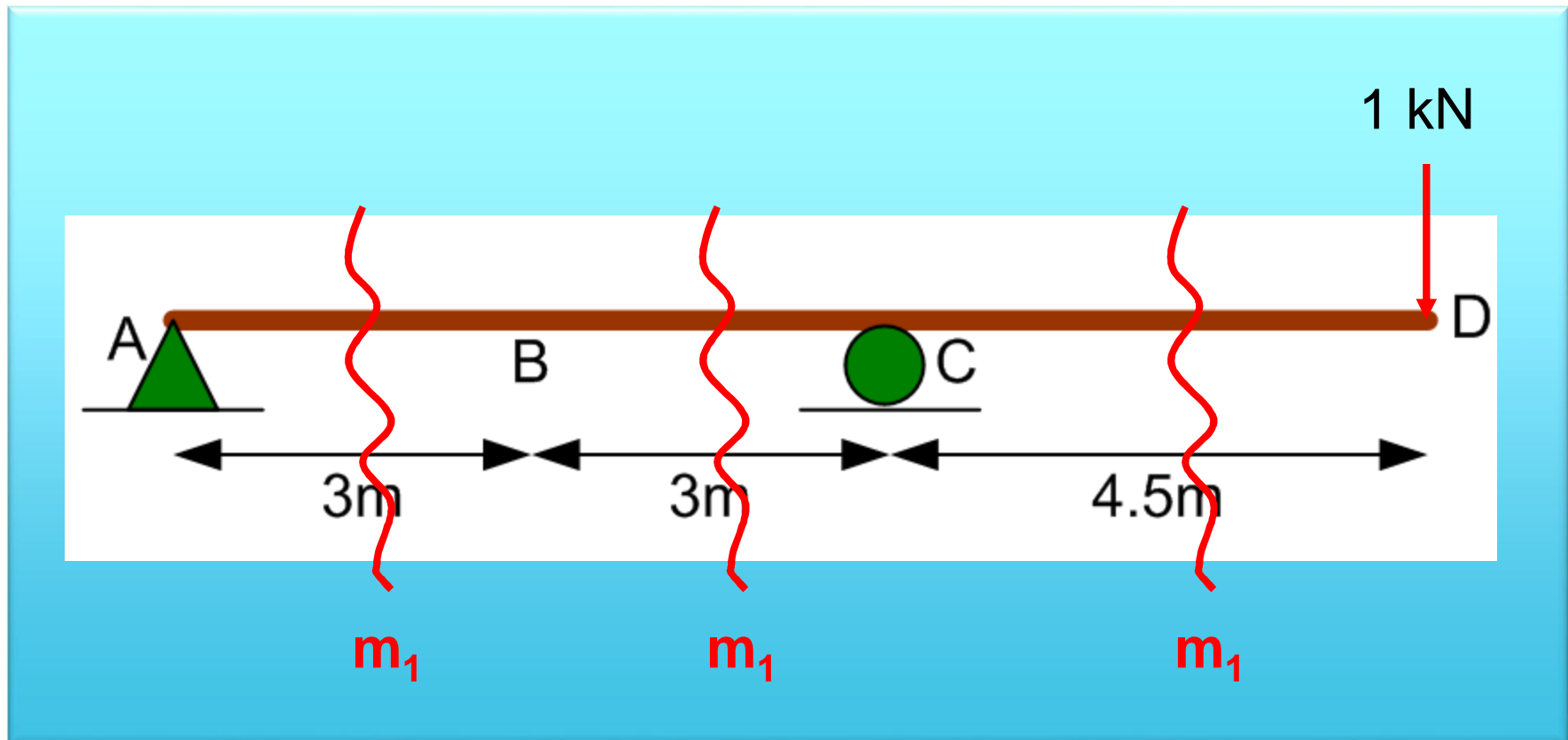
$$0 \leq x \leq 4.5$$



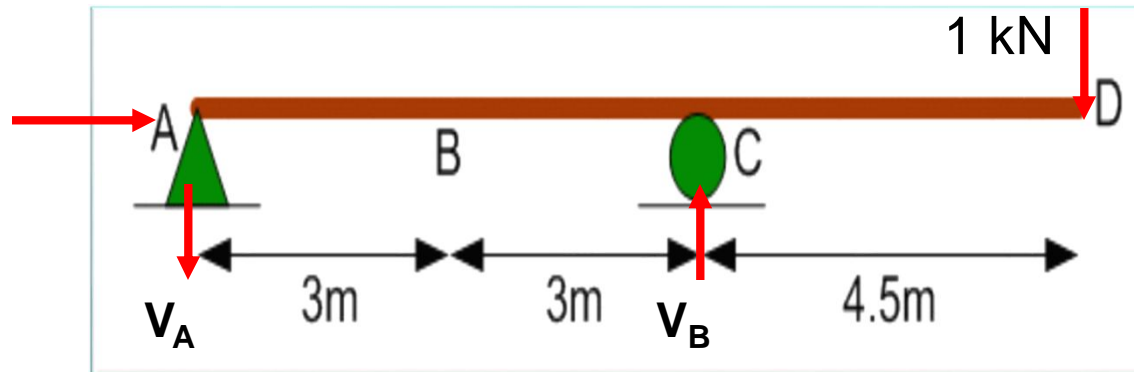
$$\sum M_x = 0 \text{ (clockwise + ve)}$$
$$M_3 = 0$$



## Virtual moment, $m$



## Determine the reaction at support



$$\sum M_A = 0 \text{ (clockwise + ve)}$$

$$1(10.5) - V_B(6) = 0$$

$$V_B = 1.75 \text{ kN}$$

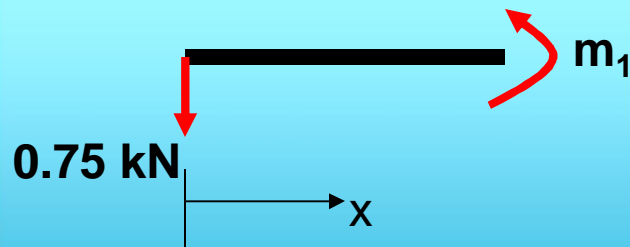
$$\sum F_y = 0 \text{ (upward + ve)}$$

$$-V_A + 1.75 - 1 = 0$$

$$V_A = 0.75 \text{ kN}$$

Member AB : LHS

$$0 \leq x \leq 3$$



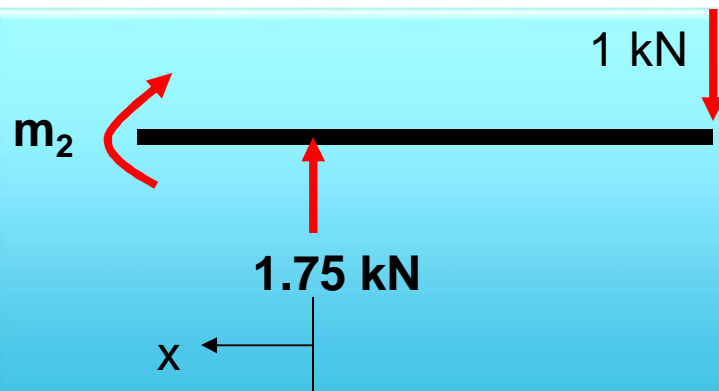
$$\sum M_x = 0 \text{ (clockwise + ve)}$$

$$-m_1 - 0.75(x) = 0$$

$$m_1 = -0.75x$$

Member BC : RHS

$$0 \leq x \leq 3$$



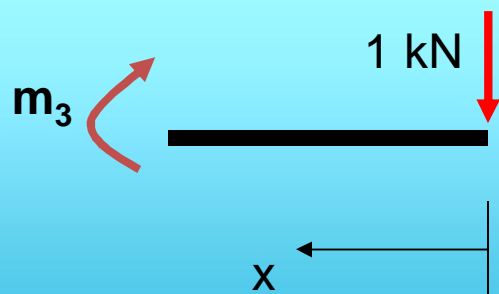
$$\sum M_x = 0 \text{ (clockwise + ve)}$$

$$m_2 - 1.75(x) + 1(x + 4.5) = 0$$

$$m_2 = 0.75(x) - 4.5$$

Member CD : RHS

$$0 \leq x \leq 4.5$$



$$\sum M_x = 0 \text{ (clockwise + ve)}$$

$$m_3 + 1 \cdot x = 0$$

$$m_3 = -x$$

## Virtual-Work Equation

$$\begin{aligned}
 1 \text{ kN} \cdot \Delta_d &= \int_0^L \frac{mM}{EI} dx \\
 &= \int_0^3 \frac{m_1 M_1}{EI} dx + \int_0^3 \frac{m_2 M_2}{EI} dx + \int_0^{4.5} \frac{m_3 M_3}{EI} dx \\
 &= \int_0^3 \frac{(-0.75x)(120 - 5x)}{EI} dx + \int_0^3 \frac{(0.75x - 4.5)(35x)}{EI} dx \\
 &\quad + \int_0^{4.5} \frac{(-x)(0)}{EI} dx
 \end{aligned}$$

$$\Delta_D = -\frac{371.25}{EI} - \frac{472.5}{EI} + \frac{0}{EI}$$

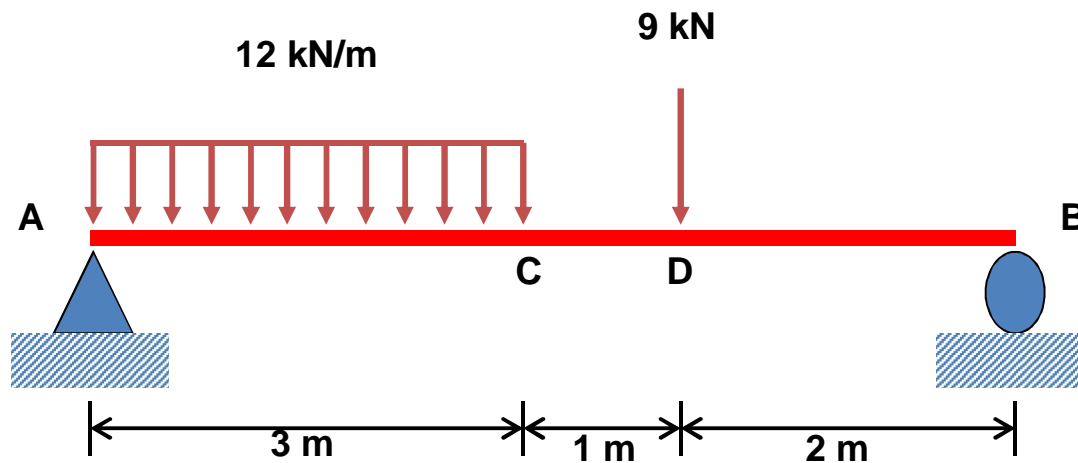
$$= -\frac{843.75 \text{ kN} \cdot \text{m}^3}{EI}$$

$$\Delta_D = \frac{-843.75 \text{ kN} \cdot \text{m}^3}{200 (10^6) \text{ kN} / \text{m}^2 (300 (10^6) \text{ mm}^4) (10^{-12} \text{ m}^4 / \text{mm}^4)}$$

$$= -0.0141 \text{ m} = -14.1 \text{ mm}$$

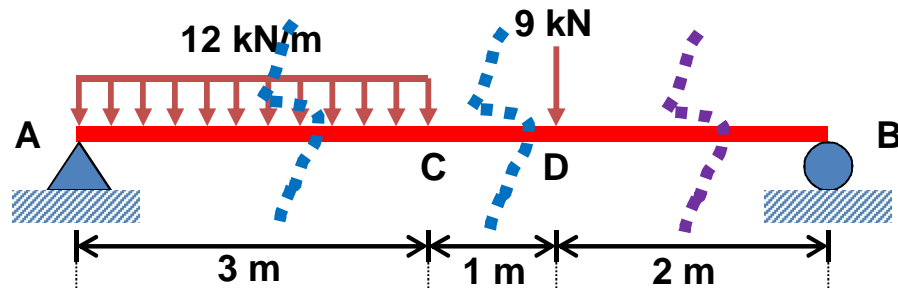
## Example 3

Determine the **slope** at A and **deflection** at C in the beam shown below

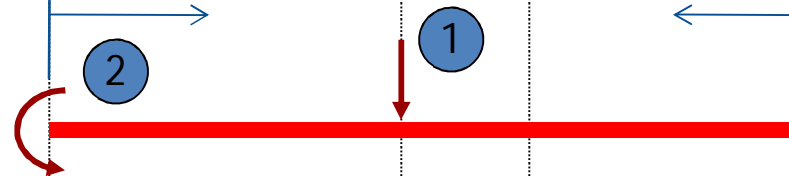


# Solution

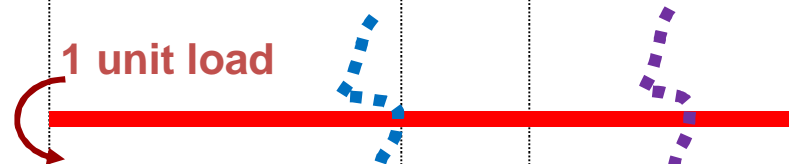
Real Load ( $M$ )



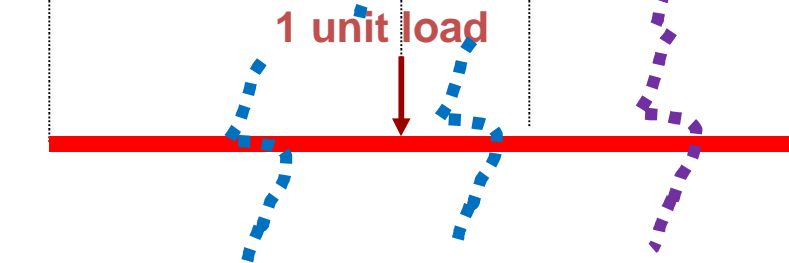
Generalized coordinates



Virtual Load ( $m_\theta$ ): Slope



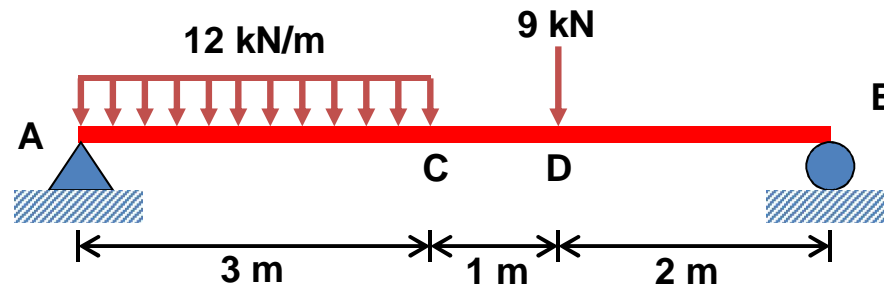
Virtual Load ( $m_\Delta$ ): Deflection





# Real Load $\rightarrow$ M ?

## 1. Support reaction,



$$\sum M_A = 0 (\text{clockwise } +),$$

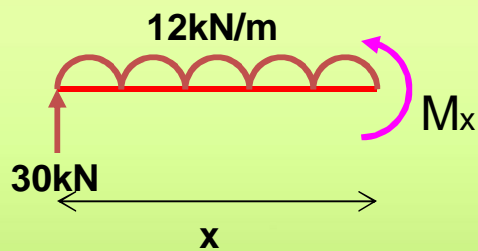
$$-R_B (6) + 9(4) + 12(3) \left( \frac{3}{2} \right) = 0$$

$$\therefore R_B = 15 \text{ kN}$$

$$\sum F_y \uparrow^+ = 0,$$

$$R_A - 12(3) - 9 + 15 = 0$$

$$\therefore R_A = 30 \text{ kN}$$

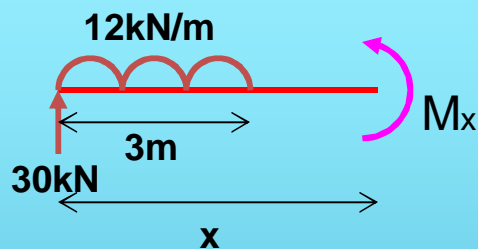


Real Load (M) :  $0 \leq x \leq 3$  (segment AC)

$$\sum M_x = 0 \text{ (clockwise +),}$$

$$M_x = 30x - 12 \left( \frac{x^2}{2} \right)$$

$$\therefore M_x = 30x - 6x^2 \dots\dots\dots(i)$$



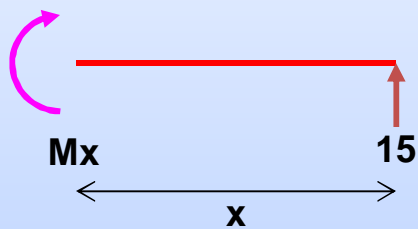
Real Load (M) :  $3 \leq x \leq 4$  (*segment CD*)

$$\sum M_x = 0 \text{ (clockwise +),}$$

$$M_x = 30x - 12(3)\left(x - \frac{3}{2}\right)$$

$$M_x = 30x - 36x + 54$$

$$\therefore M_x = -6x + 54 \dots\dots\dots(ii)$$



Real Load ( $M$ ) :  $0 \leq x \leq 2$  (*segment BD*)

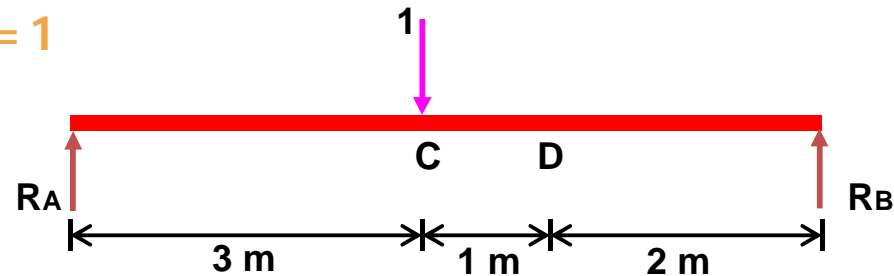
$$\sum M_x = 0 \text{ (clockwise +),}$$

$$-M_x = -15x$$

$$\therefore M_x = 15x \text{ .....(iii)}$$

# Virtual Load, m for deflection

Apply point load  $P=1$



$$\sum M_A = 0 (\text{clockwise } +),$$

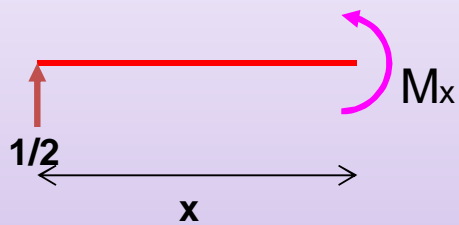
$$-R_B(6) + 1(3) = 0$$

$$\therefore R_B = \frac{1}{2}$$

$$\sum F_y \uparrow^+ = 0,$$

$$R_A + R_B - 1 = 0$$

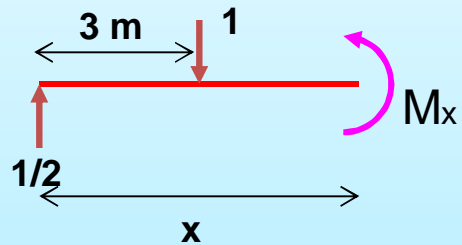
$$\therefore R_A = \frac{1}{2}$$



Virtual Load (m) :  $0 \leq x \leq 3$  (segment AC)

$$\sum M_x = 0 \text{ (clockwise +),}$$

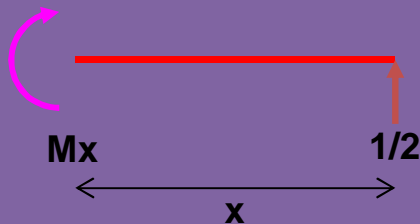
$$\therefore M_x = \frac{1}{2}x \dots \dots \dots (i)$$



Virtual Load (m) :  $3 \leq x \leq 4$  (segment *CD*)

$$\sum M_x = 0 \text{ (clockwise +),}$$

$$\therefore M_x = -0.5x + 3 \dots \dots \dots (ii)$$



Virtual Load (m) :  $0 \leq x \leq 2$  (segment *BD*)

$$\sum M_x = 0 \text{ (clockwise +),}$$

$$-M_x = -\frac{1}{2}x$$

$$\therefore M_x = \frac{1}{2}x \dots \dots \dots \text{(iii)}$$

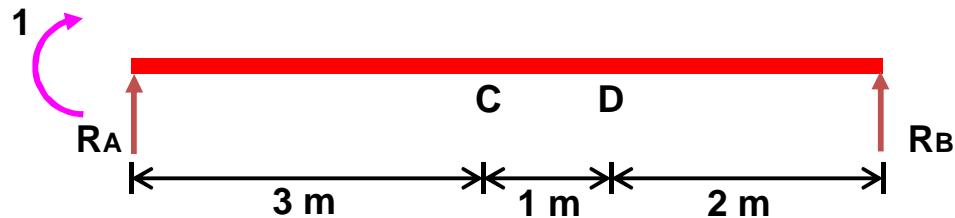


*Deflection at C,  $\Delta_D$  :*

$$\begin{aligned}\Delta_D &= \int \frac{Mm}{EI} dx \\ &= \frac{1}{EI} \int_0^3 (30x - 6x^2)(0.5x) dx + \frac{1}{EI} \int_3^4 (-6x + 54)(-0.5x + 3) dx \\ &\quad \frac{1}{EI} \int_0^2 (15x)(0.5x) dx \\ &= \frac{135.75}{EI}\end{aligned}$$

# Virtual Load, $m$ for rotation

Apply  $m_{\theta} = 1$



$$\sum M_A = 0 (\text{clockwise } +),$$

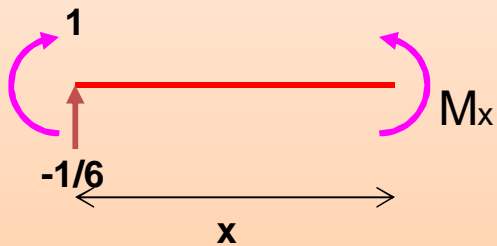
$$-R_B(6) + 1 = 0$$

$$\therefore R_B = \frac{1}{6}$$

$$\sum F_y \uparrow^+ = 0,$$

$$R_A + R_B = 0$$

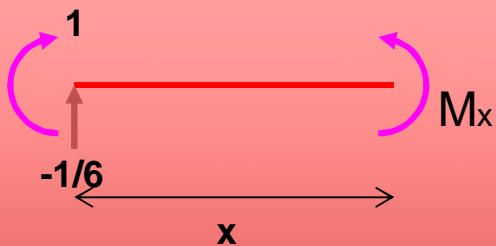
$$\therefore R_A = -\frac{1}{6}$$



Virtual Load (m) :  $0 \leq x \leq 3$  (segment AC)

$$\sum M_x = 0 \text{ (clockwise +),}$$

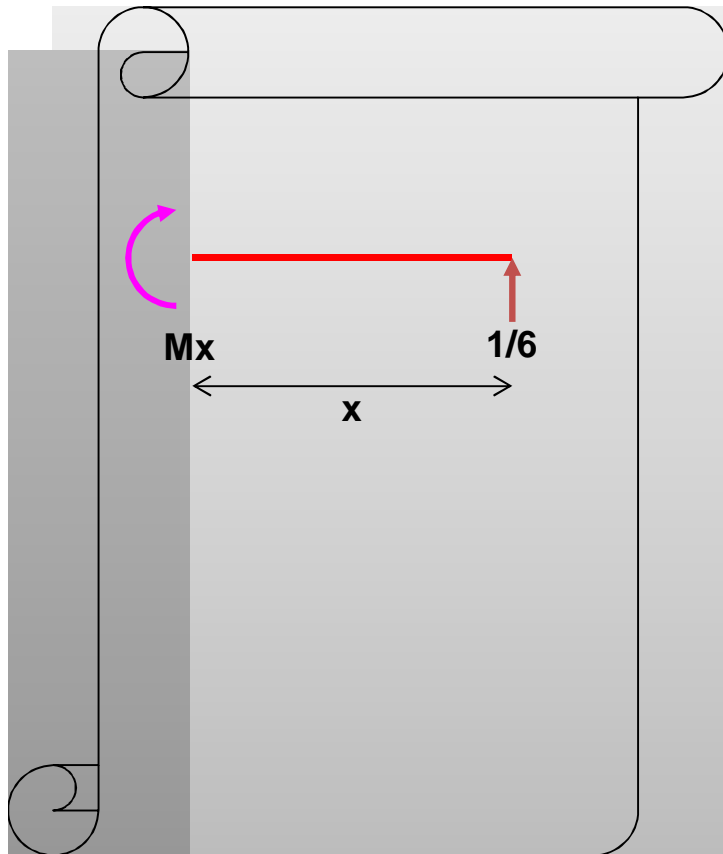
$$\therefore M_x = 1 - \frac{1}{6}x \dots\dots\dots(i)$$



Virtual Load (m) :  $3 \leq x \leq 4$  (*segment CD*)

$$\sum M_x = 0 \text{ (clockwise +),}$$

$$\therefore M_x = 1 - \frac{1}{6}x \dots \dots \dots (ii)$$



Virtual Load (m) :  $0 \leq x \leq 2$  (segment *BD*)

$$\sum M_x = 0 \text{ (clockwise +),}$$

$$-M_x = -\frac{1}{6}x$$

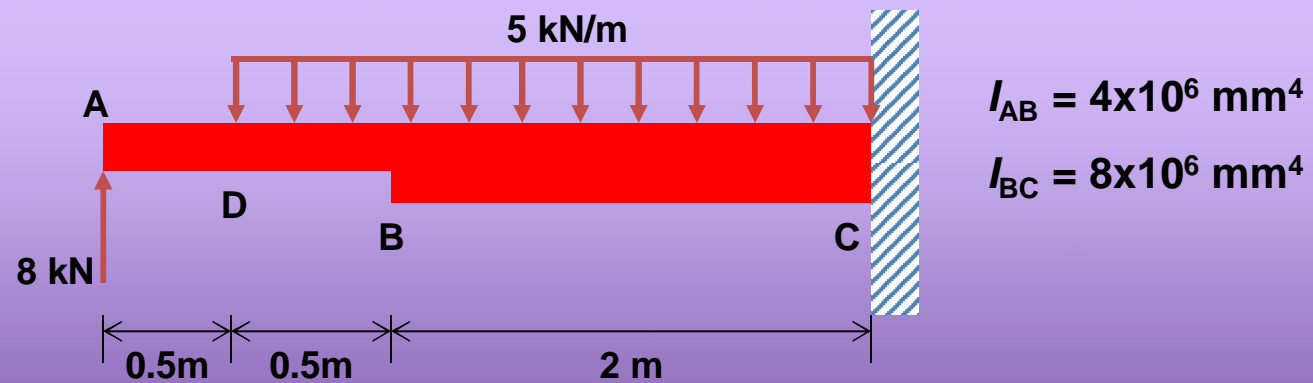
$$\therefore M_x = \frac{1}{6}x \dots \dots \dots \text{(iii)}$$

*Slope at A,  $\theta_A$  :*

$$\begin{aligned}
 \theta_A &= \int \frac{Mm}{EI} dx \\
 &= \frac{1}{EI} \int_0^3 (30x - 6x^2) \left(1 - \frac{x}{6}\right) dx + \frac{1}{EI} \int_3^4 (-6x + 54) \left(1 - \frac{x}{6}\right) dx \\
 &\quad + \frac{1}{EI} \int_0^2 (15x) \left(\frac{x}{6}\right) dx \\
 &= \frac{76.75}{EI}
 \end{aligned}$$

# Example 4

Determine the **slope and deflection at B** in the beam shown below. Given  $E=200 \text{ kN/mm}^2$



## Moment equation (deflection):

Segment	Condition	$I \text{ mm}^4$	$m \text{ (deflection)}$	$M$
AD	$0 < x < 0.5$	$4 \times 10^6$	0	$8x$
DB	$0.5 < x < 1$	$4 \times 10^6$	0	$8x - 2.5(x - 0.5)^2$
BC	$1 < x < 3$	$8 \times 10^6$	$x - 1$	$8x - 2.5(x - 0.5)^2$



*Deflection,  $\Delta_B$*

$$\begin{aligned}\Delta_B &= \int \frac{mM}{EI} dx \\ &= \int_1^3 \frac{(x-1)(-2.5x^2 + 10.5x - 0.625)}{(200 \times 10^6)(8 \times 10^{-6})} dx \\ &= \frac{1}{1600} \left[ -\frac{2.5x^4}{4} + \frac{13x^3}{3} - \frac{11.125x^2}{2} + 0.625x \right]_1^3 \\ &= 0.012m \\ &= 12mm\end{aligned}$$

## Moment equation (slope):

Segment	Condition	$I \text{ mm}^4$	$m \text{ (slope)}$	$M$
AD	$0 < x < 0.5$	$4 \times 10^6$	0	$8x$
DB	$0.5 < x < 1$	$4 \times 10^6$	0	$8x - 2.5(x - 0.5)^2$
BC	$1 < x < 3$	$8 \times 10^6$	-1	$8x - 2.5(x - 0.5)^2$

*Slope,  $\theta_B$*

$$\begin{aligned}\theta_B &= \int \frac{mM}{EI} dx \\ &= \int_1^3 \frac{(1)(-2.5x^2 + 10.5x - 0.625)}{(200 \times 10^6)(8 \times 10^{-6})} dx \\ &= \frac{1}{1600} \left[ -\frac{2.5x^3}{3} + \frac{10.5x^2}{2} - 0.625x \right]_1^3 \\ &= \frac{19.1}{1600} \\ &= 0.0119 \text{ rad}\end{aligned}$$

# THANKS



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